

# Comparability of the Non-linear and Linearized Stability Assessment During Railway Vehicle Design



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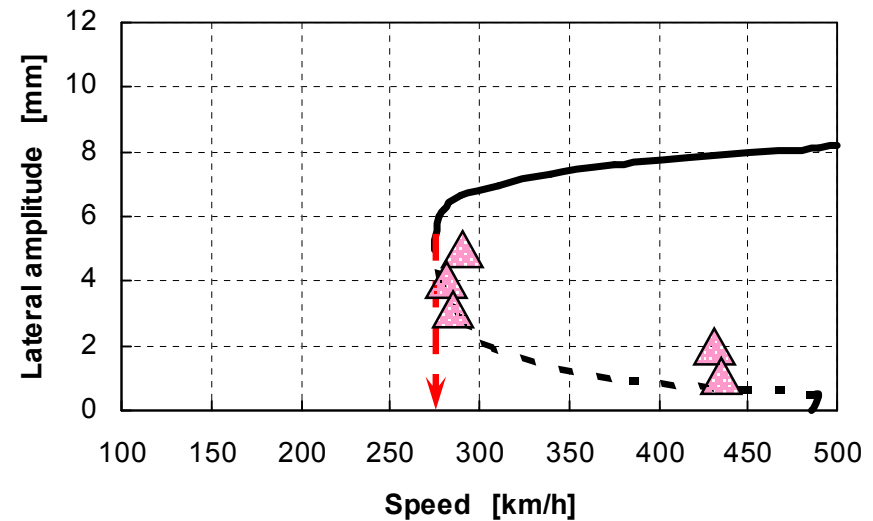
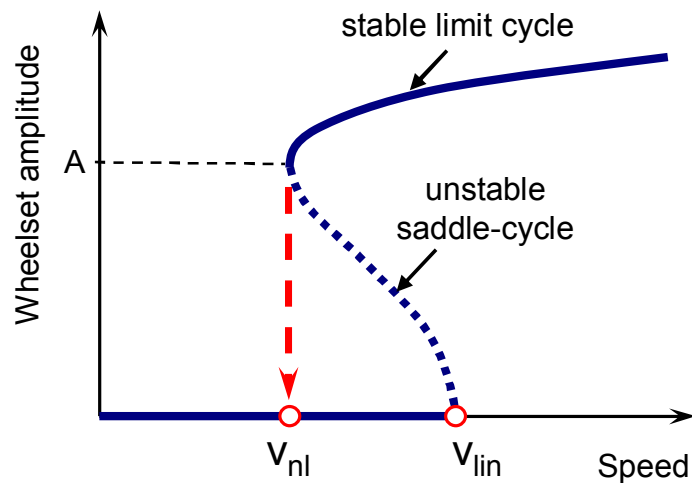
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- **Non-linear stability assessment**
- **Comparison of non-linear and linearized stability assessment**
- **Conclusions**

# Introduction

- The dynamics of the vehicle at the stability limit is nonlinear by
  - contact geometry wheelset/track
  - creep forces between wheel and rail
- Usual presentation in literature:
- Real applications in simulation tools: Quasi-linear wheel/rail contact model



- The relation between the linear and non-linear critical speed depends on the method and parameters applied

## Quasi-linear contact model wheelset/track

- Contact geometry function:

- Difference of rolling radii

$$\Delta r = r_l - r_r = f(y)$$

- Contact angle difference
- Wheelset roll angle

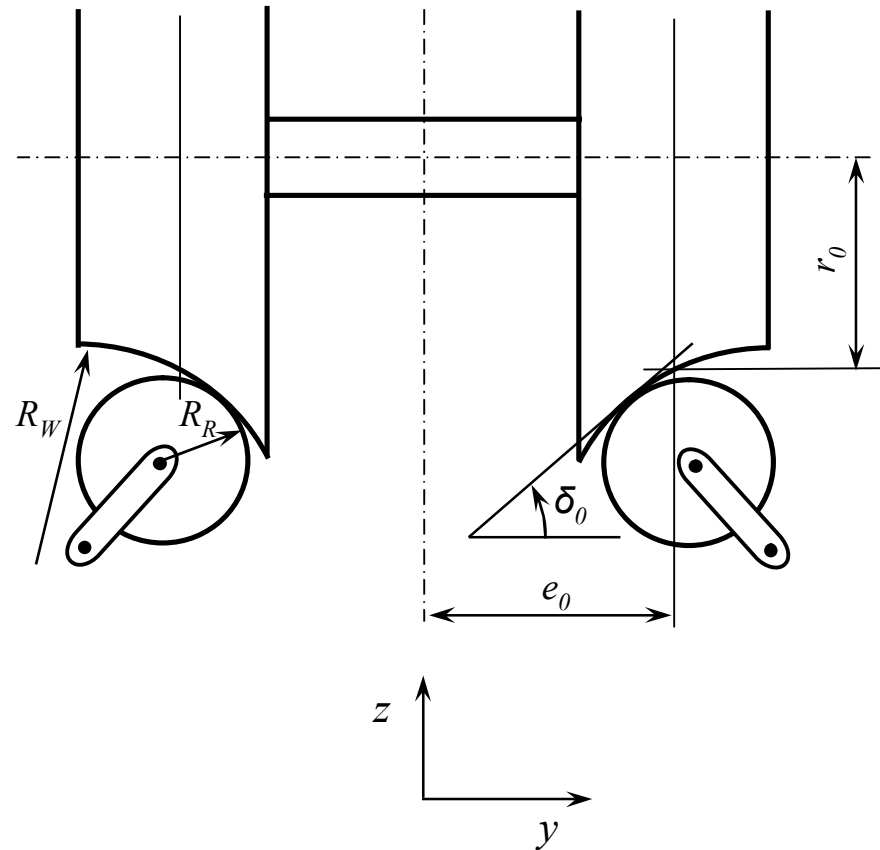
- Linearized parameters:

- Equivalent conicity as a function of wheelset amplitude  $A$

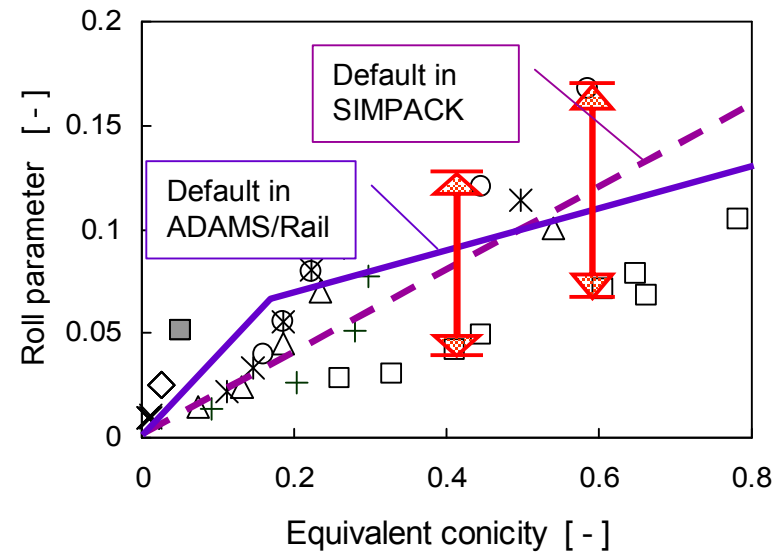
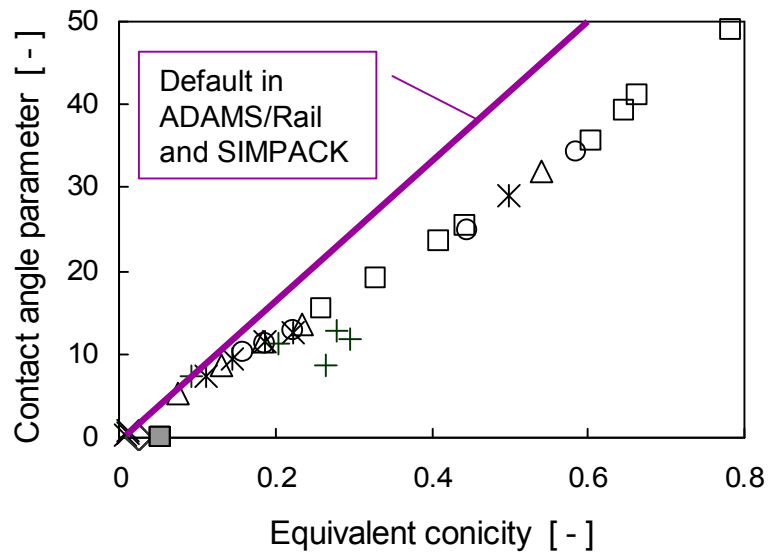
$$\lambda(A) = \frac{1}{2\pi A} \int_0^{2\pi} \Delta r(A \sin \varphi) \cdot \sin \varphi d\varphi$$

- Contact angle parameter
- Roll parameter

- Is the equivalent conicity sufficient to specify the quasi-linear contact model?



# Linearization parameters for real profile combinations

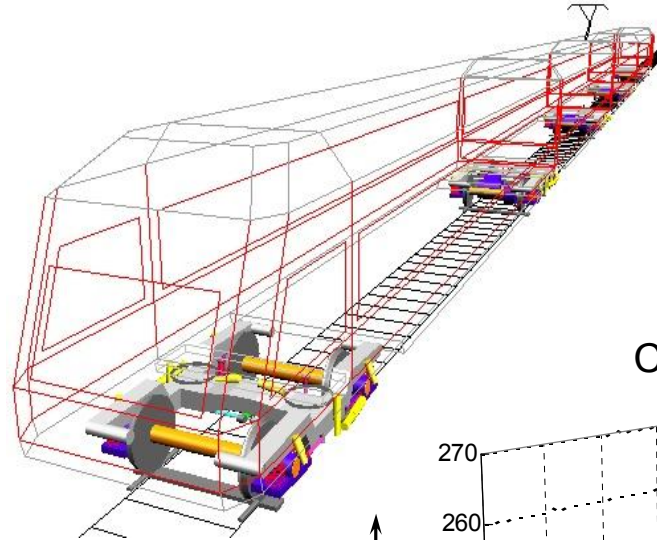


## Wheel/rail combinations :

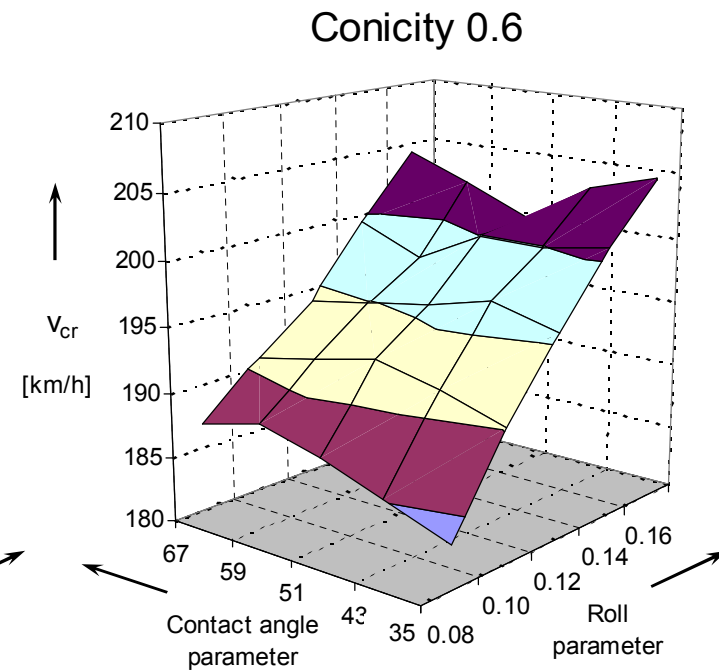
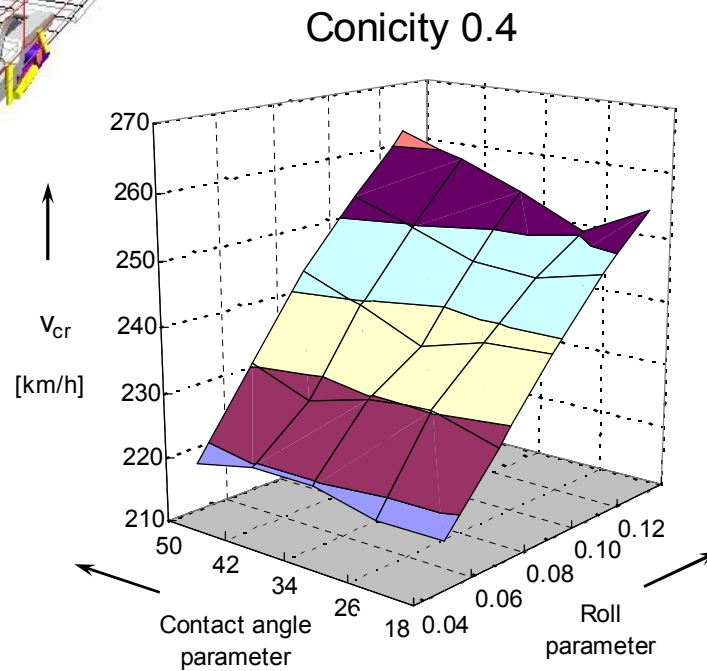
- △ S1002 / UIC60 1:40
- ✱ S1002 / UIC54 1:40
- ◇ Cone 1:40 / UIC60 1:20
- + P8 / UIC60 1:20

- × S1002 / UIC60 1:20
- S1002 / UIC54E 1:40
- Cone 1:20 / 115RE 1:40
- S1002 / UIC54E 1:40 worn crown

# Influence of contact angle and roll parameters



- The linear critical speed can differ even for the same equivalent conicity

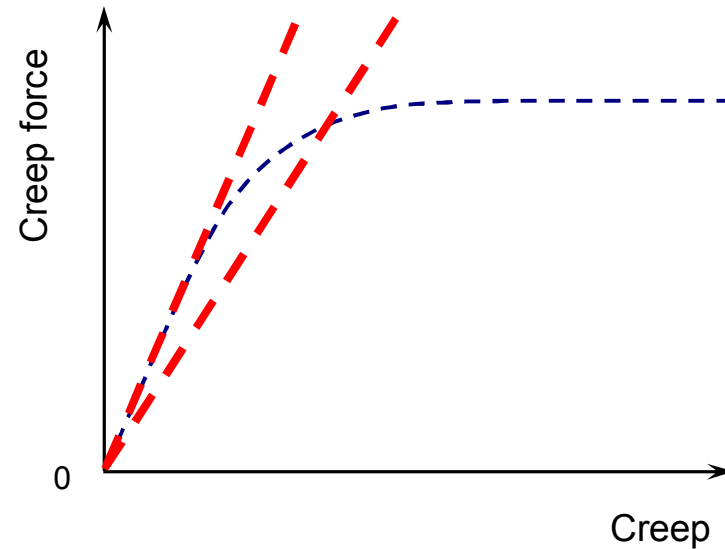


## Other parameters influencing linearized analysis

- Creep force law, represented by Kalker's factor

- Full creep: 1.0

- Reduced creep: 0.67



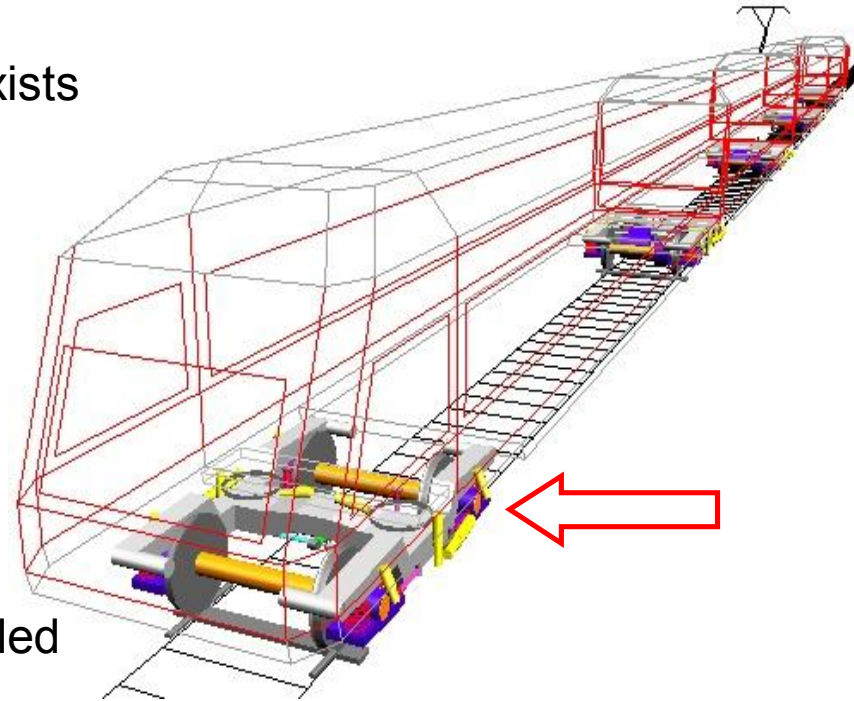
- Minimum damping to ensure stable running

- 0 %

- 5 %

# Overview of non-linear stability calculation methods

- A diversity of non-linear methods exists for stability assessment of railway vehicles dependent on
  - method
  - analysed values
  - criteria
- A four-car articulated vehicle modelled in Simpack
- Four examples of contact geometry wheelset/track with high equivalent conicity and gauge 1435 mm
- Wheel/rail friction coefficient 0.4 (dry)
- Results for the second wheelset



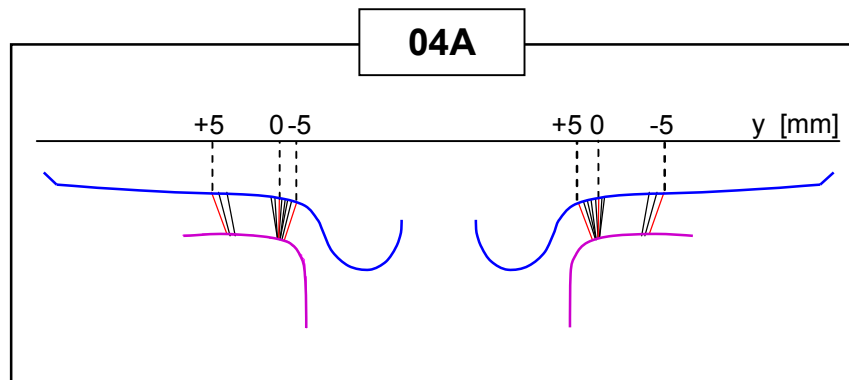
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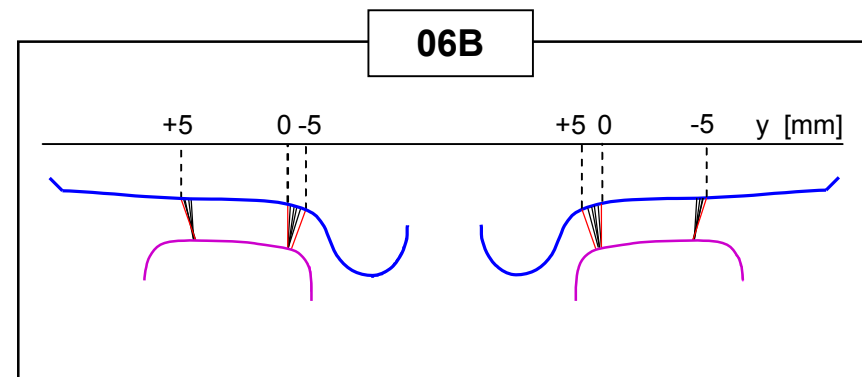
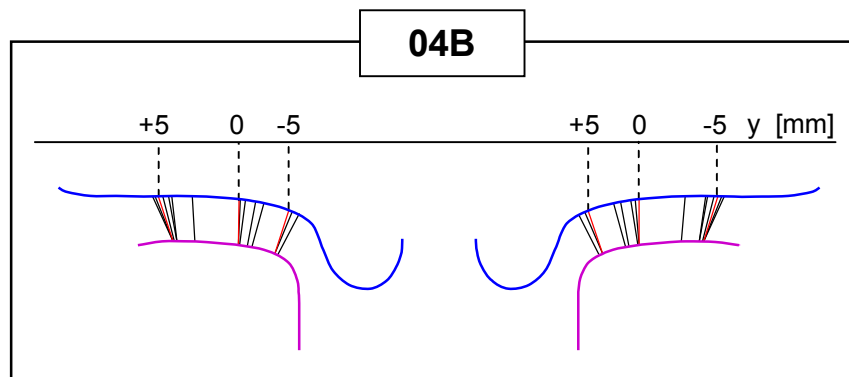
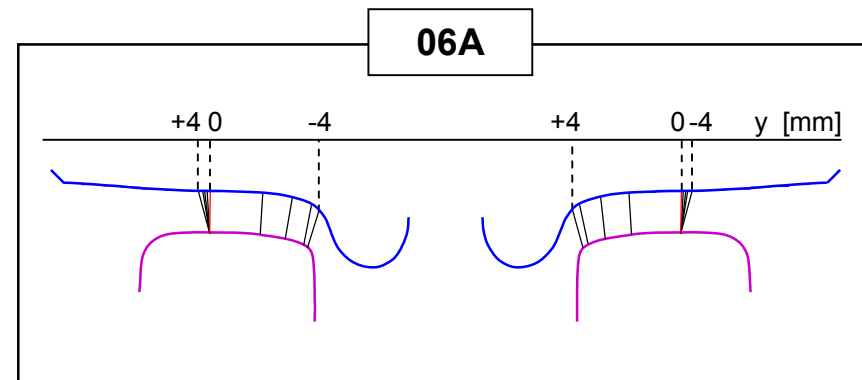
# Contact geometry wheelset/track

- In railway standards (UIC Code 518), the contact geometry wheelset/track is described by the equivalent conicity for wheelset lateral amplitude of 3 mm

Equivalent conicity 0.4

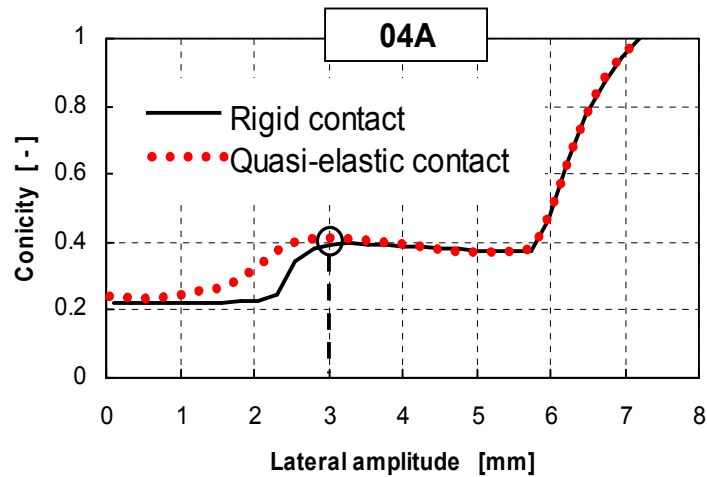


Equivalent conicity 0.6

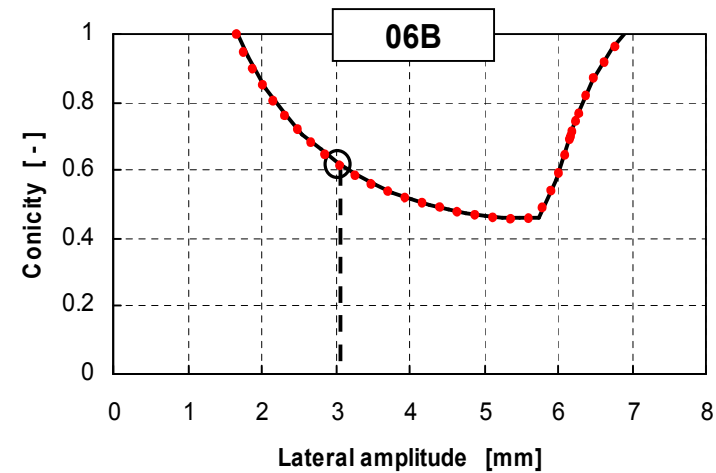
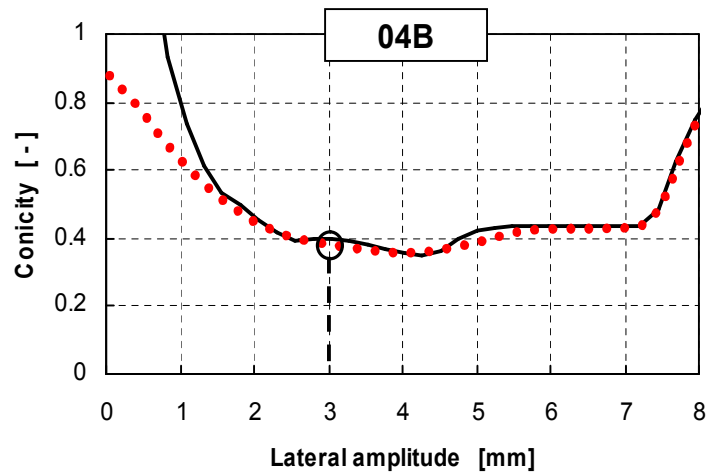
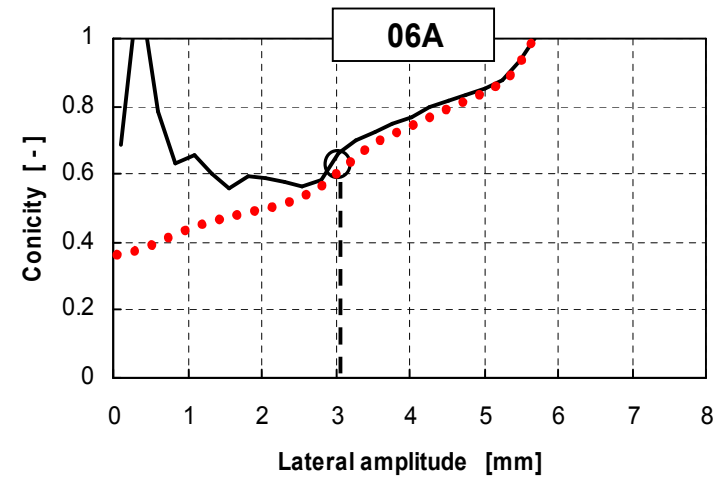


# Equivalent conicity as function of wheelset amplitude

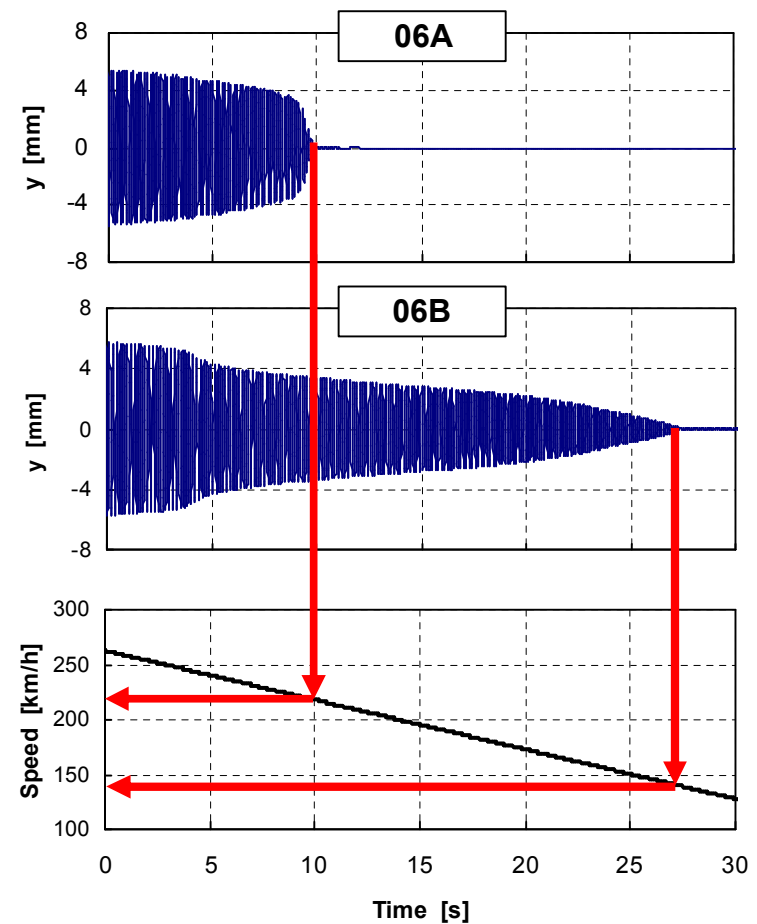
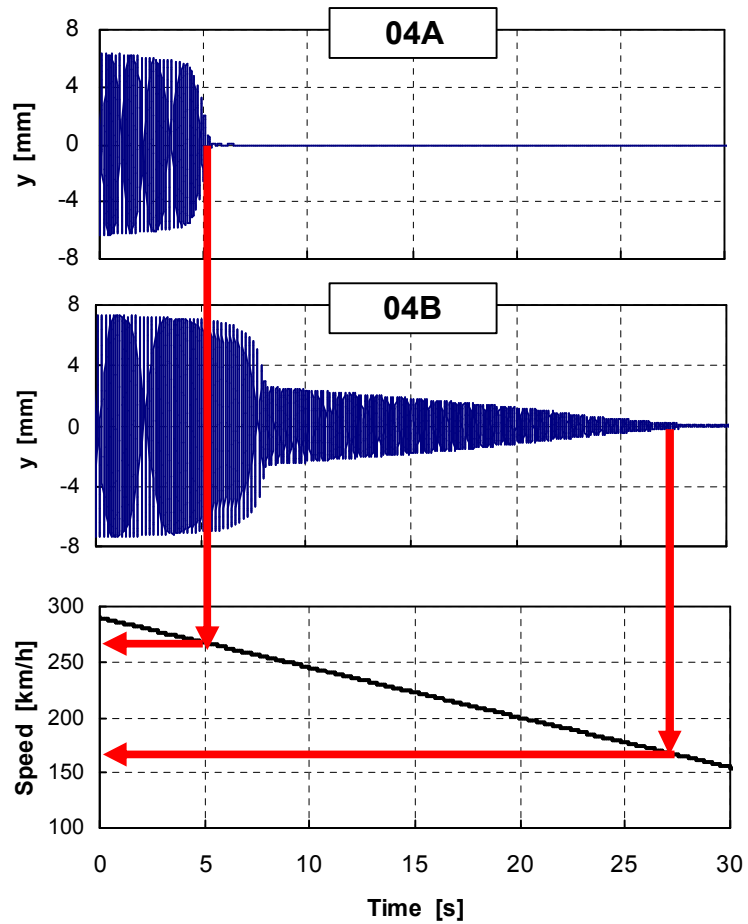
Equivalent conicity 0.4



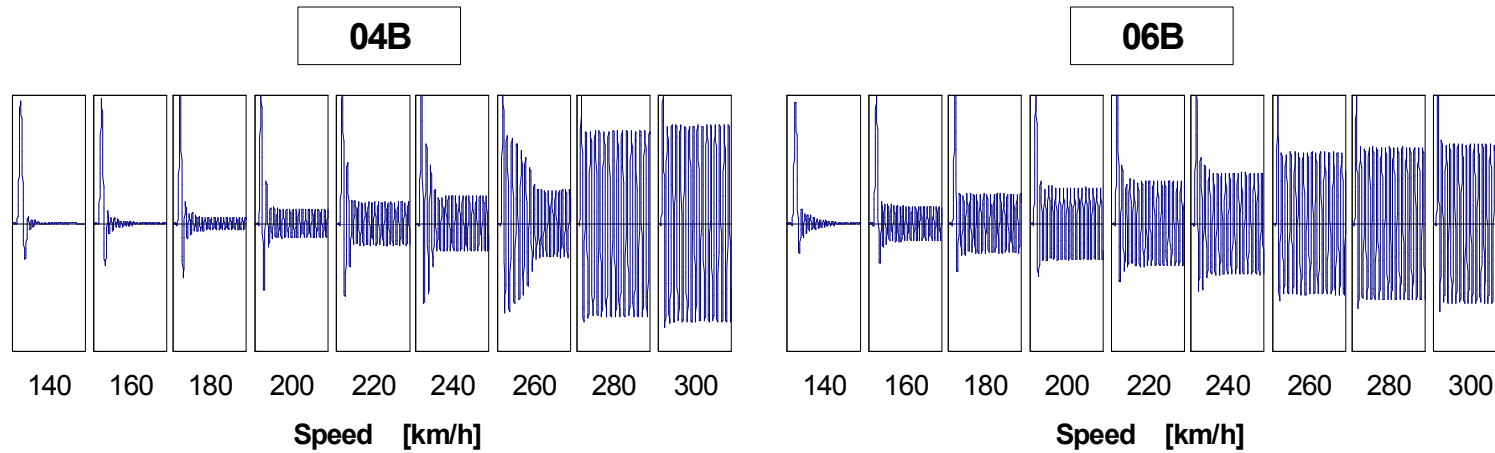
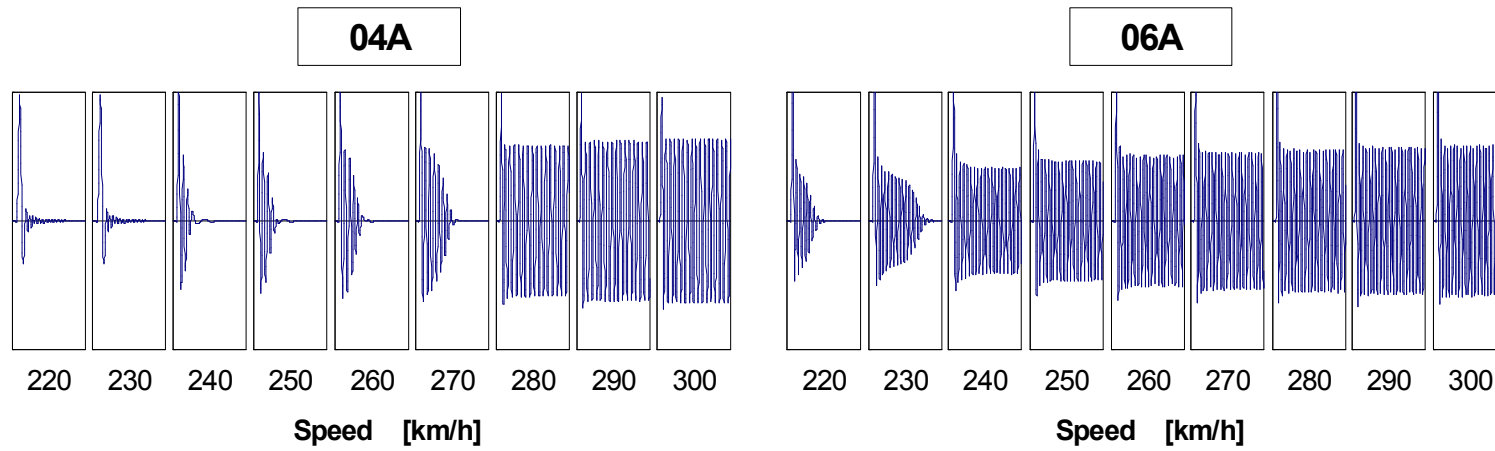
Equivalent conicity 0.6



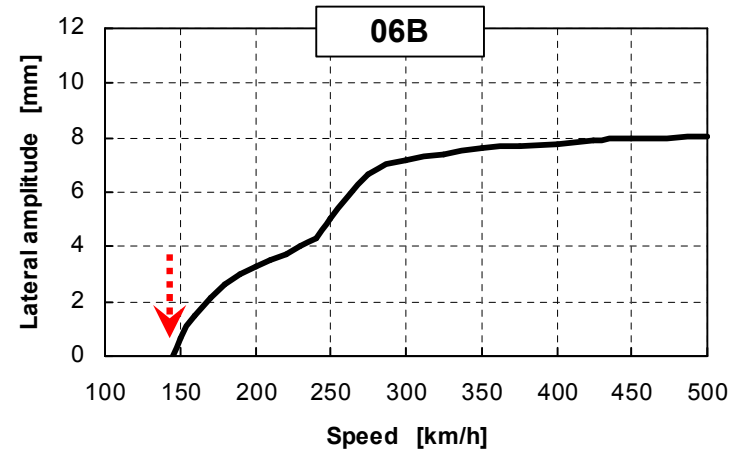
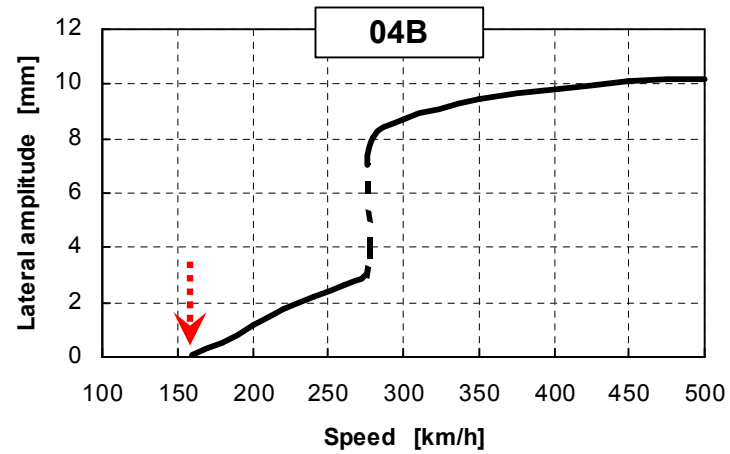
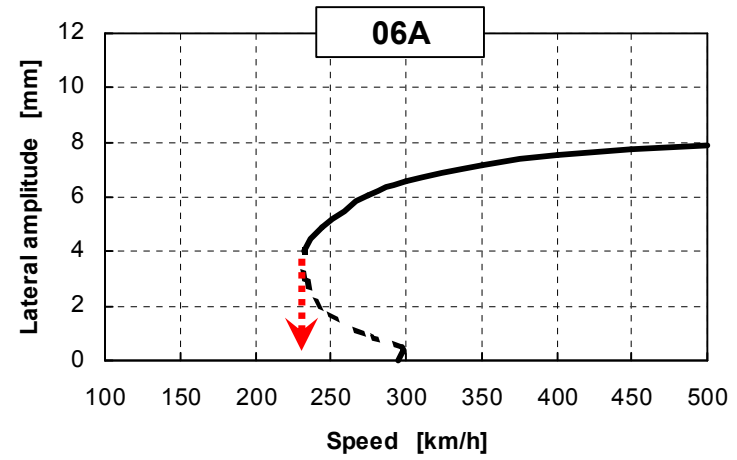
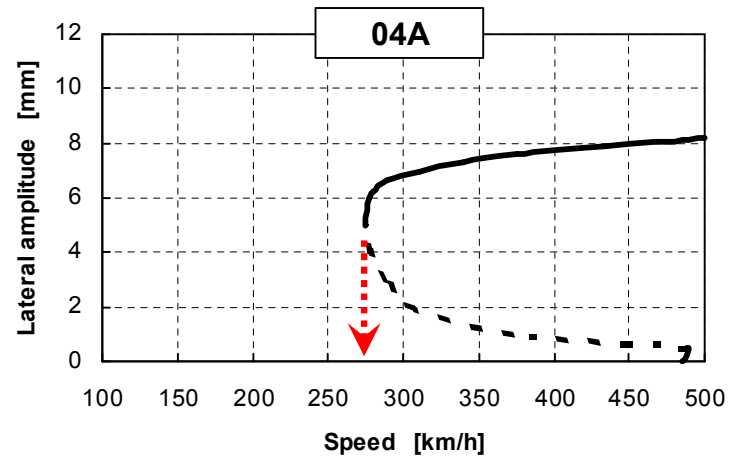
# Simulation of run with decreasing speed



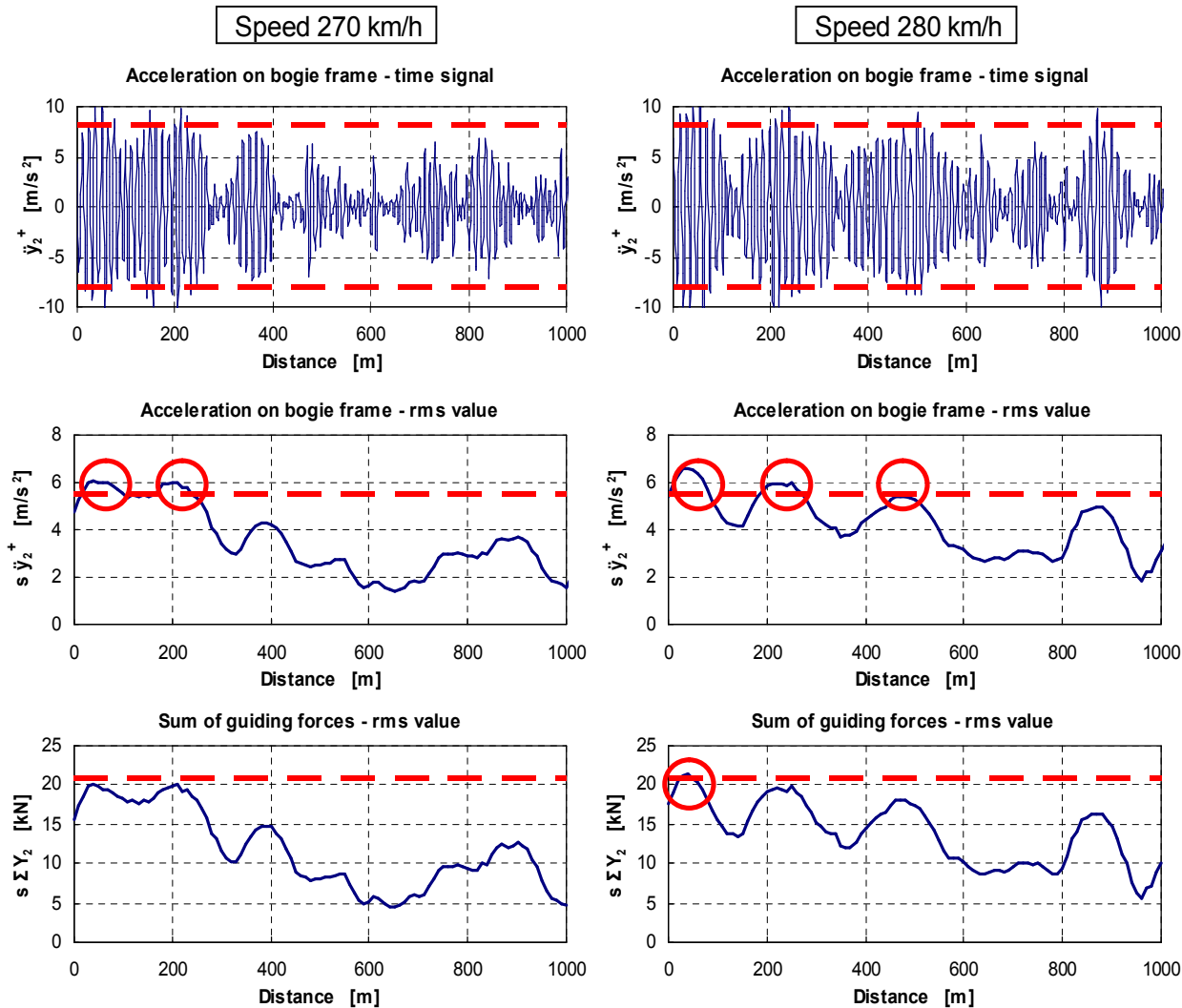
# Method with single excitation



# Bifurcation diagrams

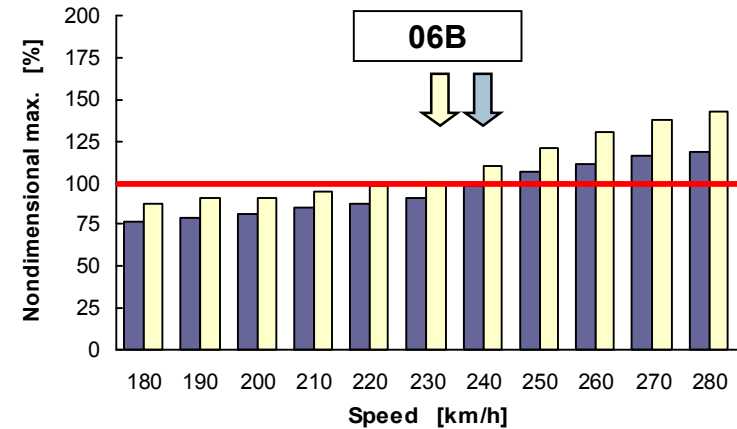
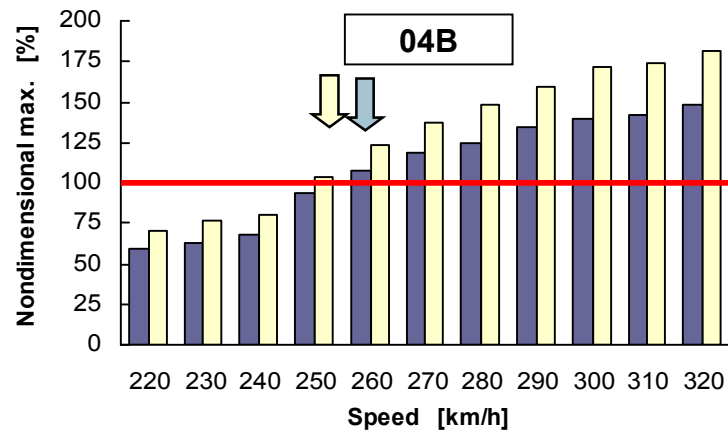
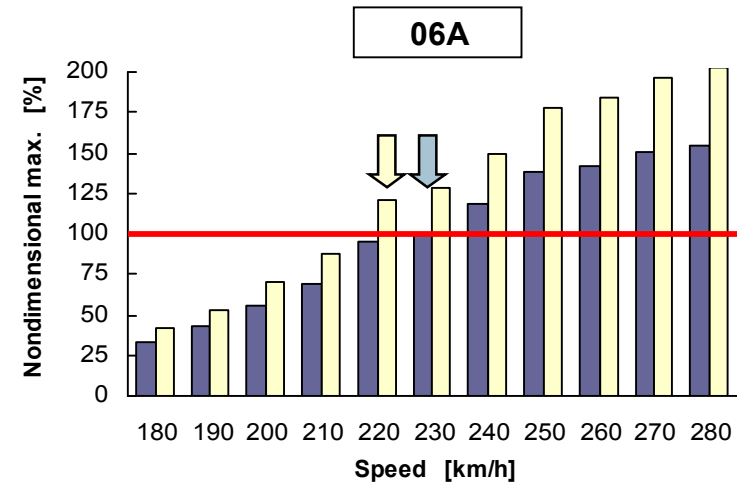
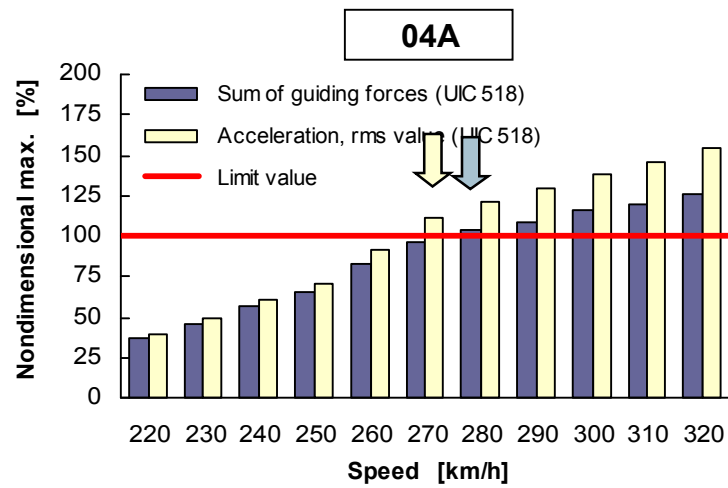


# Simulations of run on measured track irregularities

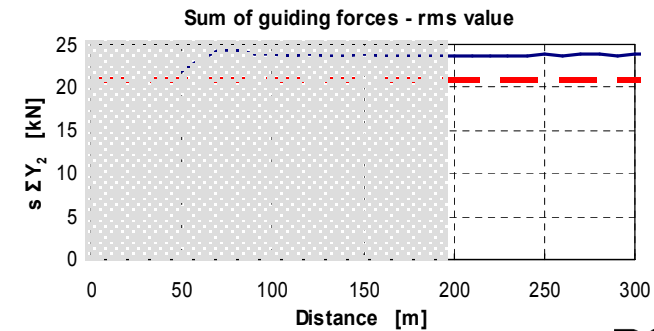
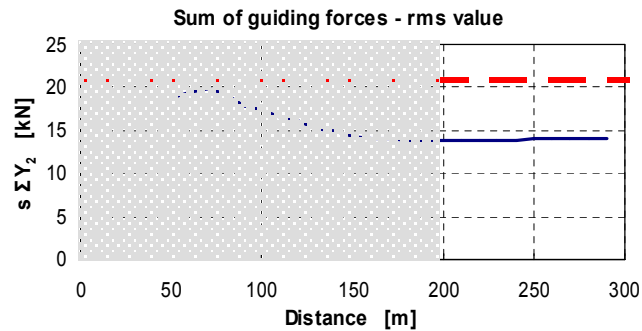
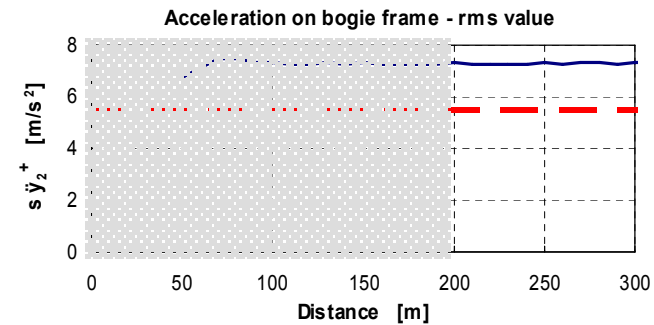
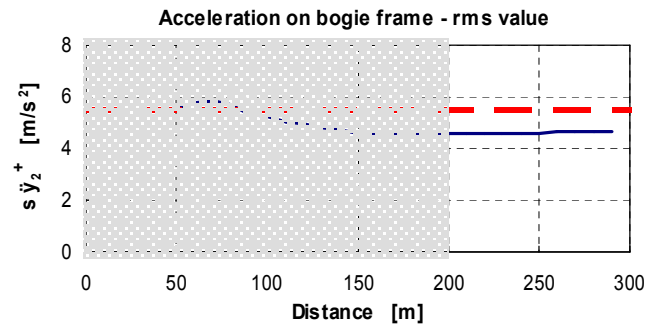
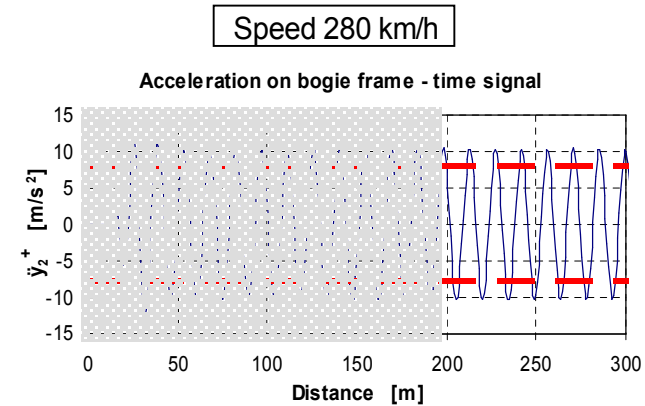
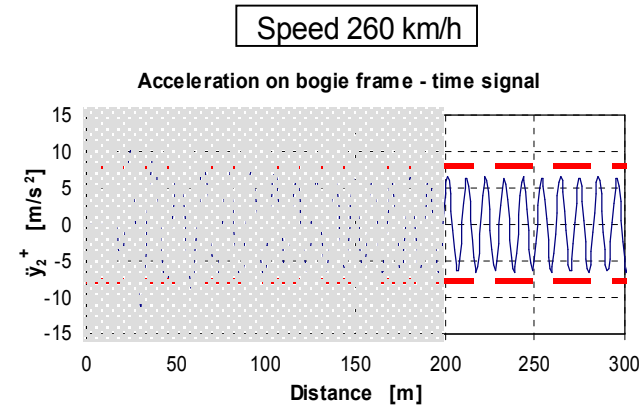


- Lateral acceleration on the bogie frame
- Lateral acceleration, sliding rms value (UIC 518)
- Sum of guiding forces, sliding rms value (UIC 518)

# Simulations of vehicle acceptance tests

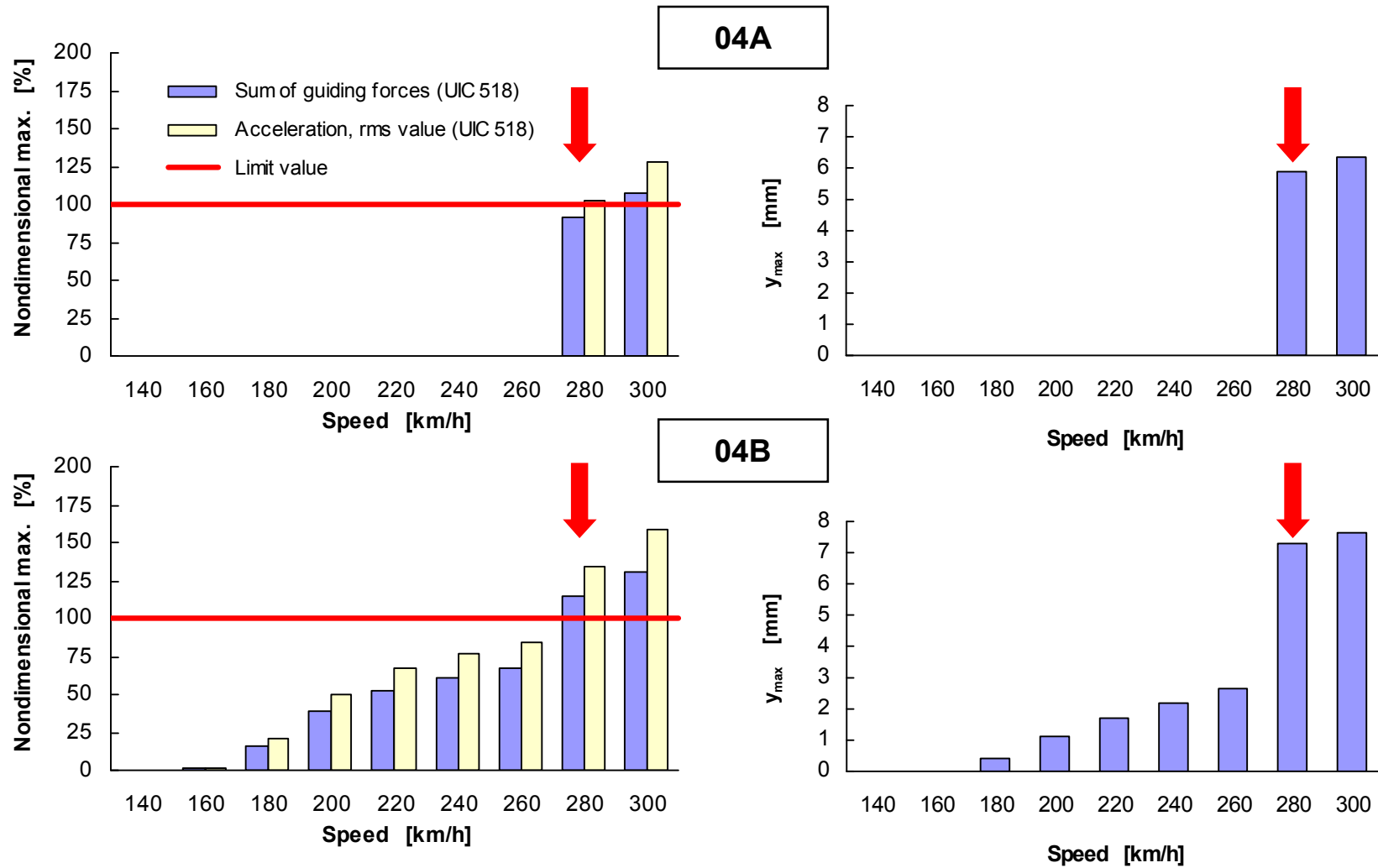


# Dynamic behaviour after a single excitation





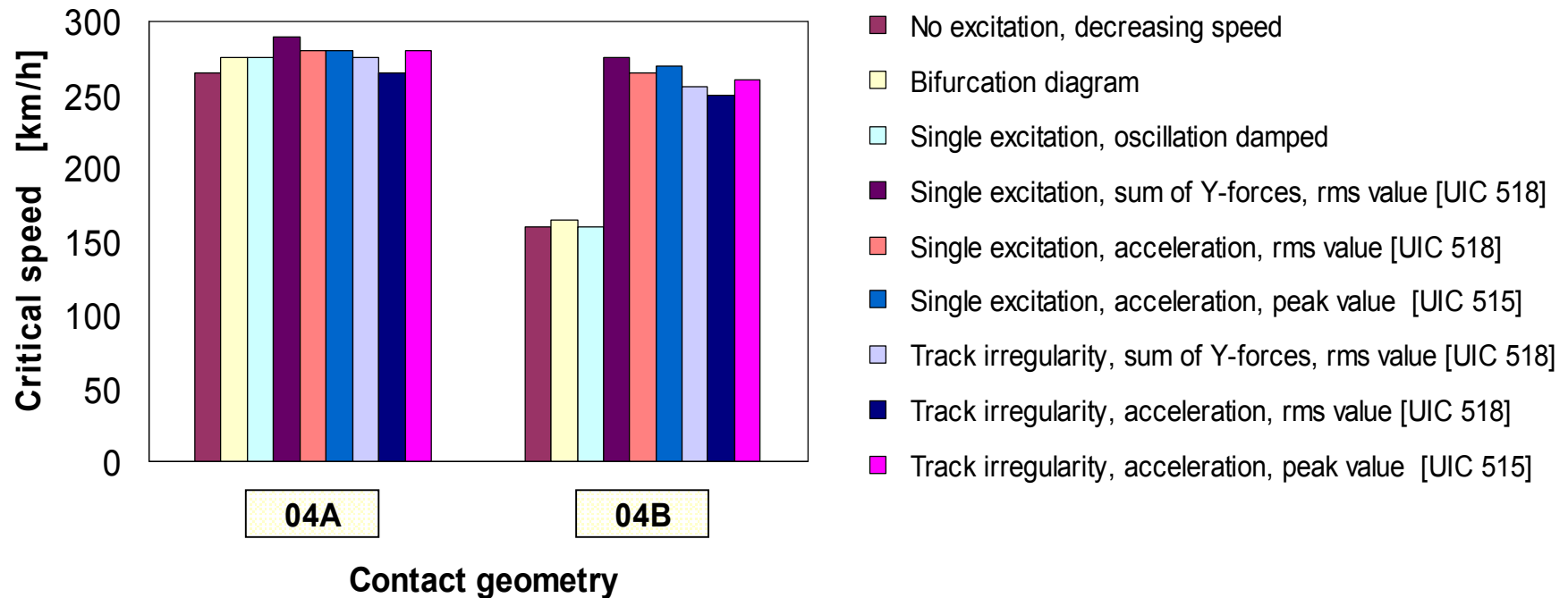
# Assessment of behaviour after a single excitation



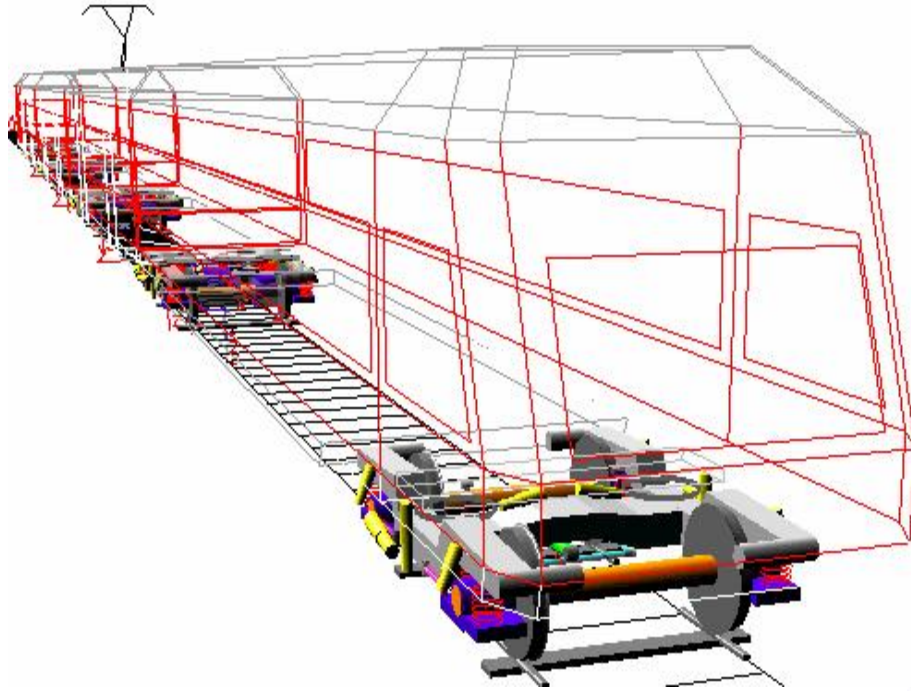
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# Comparison of resultant critical speeds

Equivalent conicity 0.4

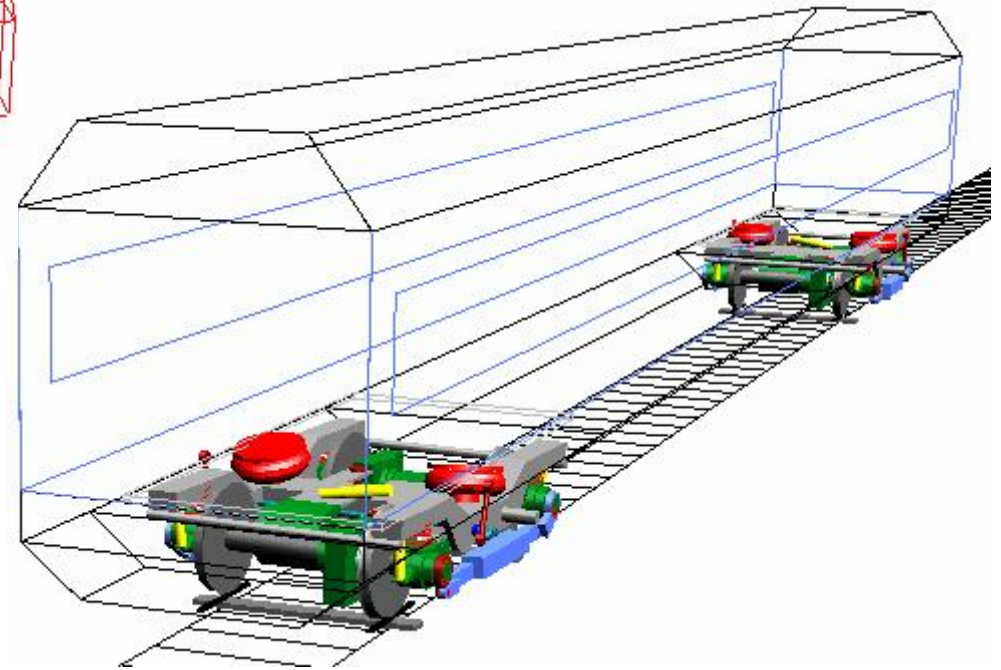


## Compared vehicle models



- Vehicle A:  
Four-car articulated vehicle  
with Jakobs' bogies and  
yaw dampers

- Vehicle B:  
Conventional passenger  
coach without yaw dampers

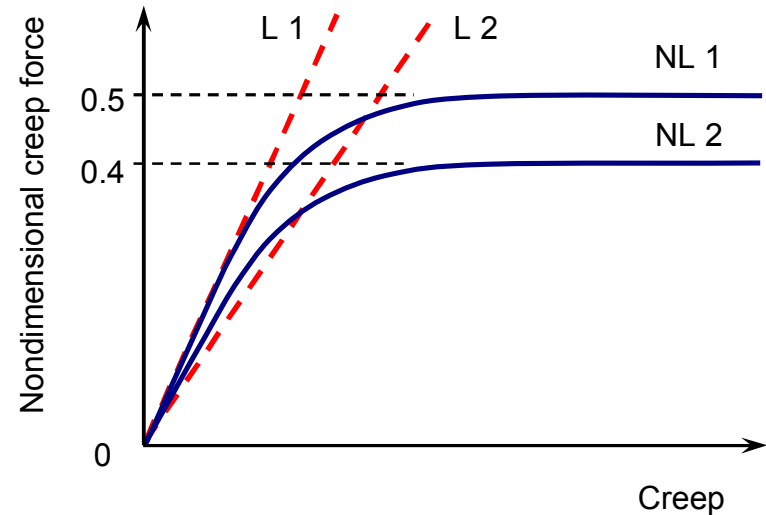


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## Compared parameters and conditions

- Quasi-linear analyses

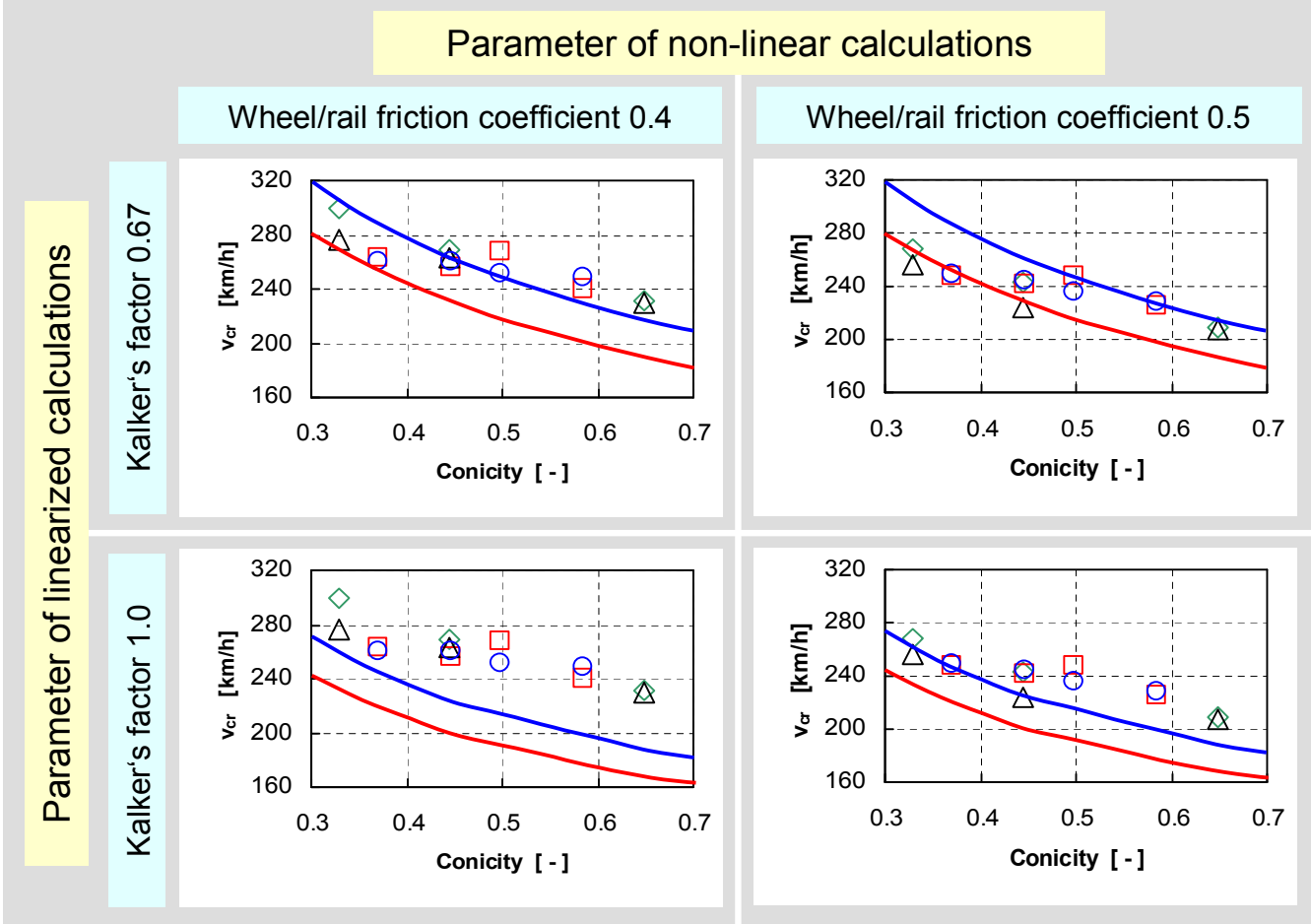
- Kalker's factor 0.67 and 1.0
- Minimum damping 0% and 5%



- Non-linear analyses

- Wheel/rail friction coefficient 0.4 and 0.5
- Wheel/rail pairings to set up the specified conicity by
  - altering the track gauge
  - wearing of the rail profile
- Method applied
  - damping behaviour after a single lateral disturbance
  - run on track with measured irregularities, criterion sum of guiding forces according to UIC 518

# Comparison of resultant critical speeds – Vehicle A



**Variation of rail profiles:**

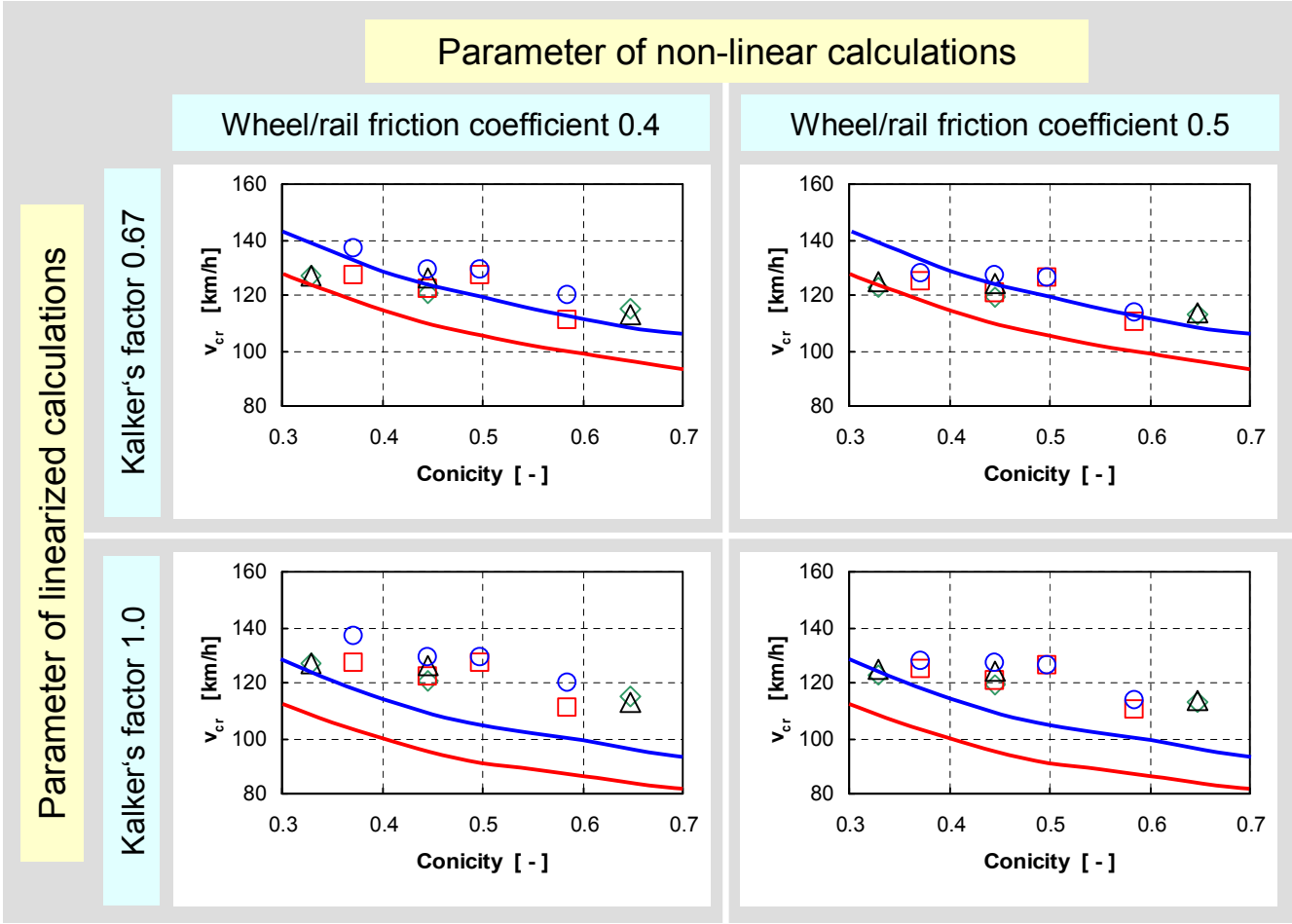
|                                 |   |  |  |
|---------------------------------|---|--|--|
| <b>Non-linear calculations:</b> | { | ◇ Criterion: Damping after a single excitation | □ Criterion: Damping after a single excitation |
|                                 |   | △ Criterion: Sum of Y-forces (UIC 518)         | ○ Criterion: Sum of Y-forces (UIC 518)         |

**Linearized calculations:**

|   |                             |
|---|-----------------------------|
| { | — (blue) Minimum damping 0% |
|   | — (red) Minimum damping 5%  |

# Comparison of resultant critical speeds – Vehicle B



|                                 |   |  |   |  |
|---------------------------------|---|--|---|--|
| <b>Non-linear calculations:</b> | } | <ul style="list-style-type: none"> <li>◇ Criterion: Damping after a single excitation</li> <li>△ Criterion: Sum of Y-forces (UIC 518)</li> </ul> | } | <ul style="list-style-type: none"> <li>□ Criterion: Damping after a single excitation</li> <li>○ Criterion: Sum of Y-forces (UIC 518)</li> </ul> |
| <b>Linearized calculations:</b> | } | <ul style="list-style-type: none"> <li>— Minimum damping 0%</li> <li>— Minimum damping 5%</li> </ul>   |   |  |

## Conclusion

- The equivalent conicity is not a sufficient input for an exact stability assessment.
- Even for the same equivalent conicity:
  - the resultant linear critical speeds can vary dependet on other linearisation parameters, Kalker's factor and residual damping,
  - the resultant nonlinear critical speeds can vary dependet on the wheel/rail contact geometry, friction coefficient, calculation method and criteria applied.
- For the application of the linearized stability calculations during vehicle design conservative parameters are recommended in order to be on the safe side in comparison to the non-linear calculations.
- It was demonstrated that, applying the minimum damping of 5% and Kalker's factor 1.0, the critical speeds calculated with quasi-linear wheel/rail contact model are below the non-linear critical speeds.